

# Speed control of two inertia system with servo motor

Yasuhide Kobayashi

December 2, 2010

## 1 Experimental setup

The control target is mainly composed of a servomotor and a inertia-load disc connected with pulleys and belt. The servo motor has a rotary encoder whose output pulse signal is processed by a counter to generate rotational position. Rotational speed is approximately measured by evaluating difference of rotational pulses. The driving torque of the servo motor is specified by analog voltage generated by D/A converter. Controllers are implemented as a real-time task of RT-Linux on PC.

Table 1: Experimental equipments

Driving motor	YASUKAWA ELEC. SGM4V-02A, SGD7V-1R6A $0.088 \times 10^{-4} \text{kgm}^2$ , 200W, 0.637Nm(rated), 1.9Nm(max), 65536ppr
PC	DELL Dimension 2100 (Celeron 1000MHz) RT-Linux 3.2 / Fedora core 1 (kernel 2.4.22)
D/A	CONTEC DA12-4(PCI) (12bit, $10\mu\text{s}$ )
Counter	CONTEC CNT24-4(PCI)H (24bit, 1MHz)

## 2 Derivation of plant model

Consider a two inertia system depicted in Fig. 1.

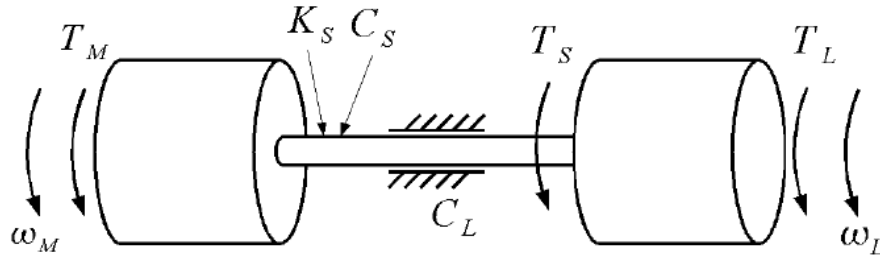


Figure 1: Two Inertia System

Denote variables as following:

- $\theta_M$  : Rotational angle of driving motor (rad)
- $\omega_M$  : Rotational speed of driving motor (rad/s)
- $J_M$  : Moment of inertia of driving motor ( $\text{Kg } m^2$ )
- $T_M$  : Driving torque (Nm)
- $\theta_L$  : Rotational angle of load (rad)
- $\omega_L$  : Rotational speed of load (rad/s)
- $J_L$  : Moment of inertia of load ( $\text{Kg } m^2$ )
- $T_L$  : Disturbance torque of load (Nm)
- $C_L$  : Damping coefficient due to friction on load
- $K_S$  : Torsional stiffness of shaft (Nm/rad)
- $C_S$  : Torsional damping coefficient of shaft
- $T_S$  : Torsional torque on shaft (Nm)

Let  $\theta_r$  be a relative angle of the motor and the load as  $\theta_r := \theta_M - \theta_L$ , then equation of motion of the system is given as following:

$$J_M \dot{\omega}_M = T_M - T_S \quad (1)$$

$$T_S = K_S \theta_r + C_S (\omega_M - \omega_L) \quad (2)$$

$$J_L \dot{\omega}_L = T_L + T_S - C_L \omega_L \quad (3)$$

Let state-space variable  $x$  be

$$x := \begin{bmatrix} \theta_r \\ \omega_M \\ \omega_L \end{bmatrix}. \quad (4)$$

State-space representation of the system from  $T_M$  to  $\omega_M$  is given by

$$\frac{d}{dt} \begin{bmatrix} \theta_r \\ \omega_M \\ \omega_L \end{bmatrix} = \begin{bmatrix} 0 & 1 & -1 \\ -\frac{K_S}{J_M} & -\frac{C_S}{J_M} & \frac{C_S}{J_M} \\ \frac{K_S}{J_L} & \frac{C_S}{J_L} & -\frac{C_S+C_L}{J_L} \end{bmatrix} \begin{bmatrix} \theta_r \\ \omega_M \\ \omega_L \end{bmatrix} + \begin{bmatrix} 0 \\ \frac{1}{J_M} \\ 0 \end{bmatrix} T_M \quad (5)$$

$$\omega_M = [0 \quad 1 \quad 0] \begin{bmatrix} \theta_r \\ \omega_M \\ \omega_L \end{bmatrix} \quad (6)$$

Torsional resonance frequency  $f_r$  and anti-resonance frequency  $f_a$  are given respectively as follows:

$$f_r = \frac{1}{2\pi} \sqrt{K_S \left( \frac{1}{J_L} + \frac{1}{J_M} \right)}, \quad f_a = \frac{1}{2\pi} \sqrt{\frac{K_S}{J_L}} \quad (7)$$